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# DESIGN OF A CLOSED LOOP SCO2 WIND TUNNEL: NUMERICAL MODELLING OF NON-EQUILIBRIUM CONDENSATION IN A CONVERGING-DIVERGING NOZZLE

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1. Introduction

2. Thermodynamic cycle and layout of the experimental test loop

3. Description of the supersonic test section and its components

4. Validation and application of the non-equilibrium condensation model

5. Conclusions and future work



S-CO, direct cycle

- since 2012-

1,200

1,400

LNG

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#### **S-CO2 APPLICATIONS AND POWER CYCLES**

- SCO2 is compressed in the nearincompressible region (cycles C and D)
- Compactness and high efficiencies
- >> Suitable for different heat sources  $(T_{cr} = 31^{\circ}C)$
- >> Higher TIT achievable compared to steam cycles





1.000

Carnot efficiency theoretical limi

-CO<sub>2</sub> indirect cycle

Steam turbir > 100years

since 2009

thermal

800

(Invernizzi, 2013)



## TRANSCRITICAL REFRIGERATION CYCLE

- CO2 is safe (not toxic, flammable or corrosive) and economic
- Alternative to ozone-depleting and global-warming refrigerants
- Possibility of operating with heat rejection temperatures close to the critical point with pressures up to 130 bar
- Technology becoming standardized, costs are strongly decreasing



(Austin et Al., 2011)



#### THERMOPHYSICAL PROPERTIES

- >> Compressibility factor:  $Z = \frac{P}{\rho RT}$
- >> Real-gas behaviour of the fluid in the supercritical region ( $Z = 0.2 \div 0.5$ )
- Sharp variations of the fluid properties in the region near the critical point (31 °C, 73.8 bar)
- Transition from liquid-like to gas-like behaviour crossing the pseudo-boiling temperature at supercritical pressures





#### **POSSIBLE CONDENSATION IN COMPRESSORS**

- Flow acceleration on the suction side of the compressor blade may cause the nucleation of liquid droplets
- Condensation would affect the performance of the machine
- Lettieri et al. defined a criterion to determine a limit for condensation-free inlet conditions of compressors



(Baltadjiev et Al, 2015)



#### THERMODYNAMIC CYCLE AND LAYOUT

- Constraints on total required power (~500 kW) and dimensions.
- >> Research interests:
  - Heat transfer across the pseudoboiling line
  - Expansion from supercritical condition
  - Non-equilibrium condensation







#### SUPERSONIC TEST SECTION







#### **CONTRACTION ZONE: CFD RESULTS**

>> Real-gas CFD simulation carried out in Ansys Fluent

>> Unstructured mesh with ICEM CFD

>> Total pressure loss coefficient:

$$Y = \frac{P_{0,in} - P_{0,out}}{P_{0,out} - P_{out}} = \begin{cases} 0.0004 & \text{Inviscid} \\ 0.0348 & \text{Viscous} \end{cases}$$





#### **NON-EQUILIBRIUM CONDENSATION MODEL**

 $\frac{4\pi r^{*2}\sigma}{3k_BT_g}\right)$ 

roplets nucleation rate: 
$$J = \frac{1}{1+\theta} \frac{\rho_g^2}{\rho_l} \sqrt{\frac{2\sigma}{\pi M^3} \cdot \exp(\frac{2\sigma}{m^3})}$$

Critical radius: 
$$r^* = \frac{2\sigma}{\rho_l R T_g \int_{P_s}^{P} \frac{Z}{P} dP} = \frac{2\sigma}{\rho_l R T_g [\ln S + A(P_s) \cdot P_s(S-1)]}$$

dr

Droplet growth rate model:

D

Droplet growth rate model: 
$$\frac{dr}{dt} = \frac{k_g}{r\rho_l \left[\frac{1}{1+4Kn} + 3.78(1-\nu)\frac{Kn}{Pr_g}\right]} \left(\frac{T_l - T_g}{L}\right)$$
$$T_l = T_s(P) - \left[T_s(P) - T_g\right]\frac{r^*}{r}$$

 $k_{q}$ 



- Non-isothermal correction term  $\theta$
- Surface tension  $\sigma$
- М Molecular mass
- Boltzmann constant  $k_B$
- Supersaturation ratio S
- Young modification parameter ν
- Kn Knudsen number
- Pr Prandtl number



#### VALIDATION OF THE CONDENSATION MODEL







#### **DESIGNED NOZZLE: EXPANSION AND CONDENSATION**

- >> Expansion rate:  $\dot{p} = -\frac{w}{p}\frac{dp}{dx}$
- >> Average value of  $2.1 \cdot 10^3 s^{-1}$  within the metastable region
- Pressure and temperature return to saturated conditions once that the Wilson line is reached







#### **DESIGNED NOZZLE: CFD RESULTS**







## **CONCLUSIONS AND FUTURE WORK**

- The sCO2 experimental test loop, which will allow the study of relevant phenomena taking place in the nonideal gas region, is presented. The loop is under commissioning.
- >> The results from the real-gas CFD simulation of the contraction zone indicate that no flow separation occurs, and a two-dimensional flow is provided to the supersonic test section.
- >> The presented condensation model is validated and applied to the designed nozzle. The model assume a linear relation between the compressibility factor and pressure around the saturation pressure.
- >> The effects of the low surface tension on condensation can be further studied and new condensation models can be developed considering the gas as non-ideal, i.e. compressibility factor lower than 1.





- 1. Invernizzi, C.M. (2013). Closed Power Cycles: Thermodynamic Fundamentals and Applications. Lectures Notes in Energy, vol. 11.
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- 3. Austin, B.T. and Sumathy, K. (2011). Transcritical carbon dioxide heat pump systems: A review. Renewable and Sustainable Energy Reviews, 15, pp. 4013-4029.
- 4. Baltadjiev, N., Lettieri, C. and Spakovszky, Z. (2015). An Investigation of Real Gas Effects in Supercritical CO2 Centrifugal Compressors. ASME Journal of Turbomachinery, 137(9), 091003.

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#### **DESIGNED NOZZLE: CFD RESULTS**

